

3.7 Mechanical and Electrical Vibrations

I. Mass-spring systems without damping

Hook's Law: $F = -kx$

relaxed
spring
 $F = 0$
 $x = 0$

stretched
spring
 $x > 0, F = -kx$
←

compressed spring
 $x < 0, F = -kx$
→

k is called the spring constant.

Units for k might be Newtons/meter = $\frac{\text{kg}}{\text{m} \cdot \text{s}^2}$
or pounds/foot = $\text{slugs/ft} \cdot \text{s}^2$.

Next we attach one end of our spring to the ceiling. Then we will attach an object of mass m to the lower end of the spring. Then we will set this system into motion.

relaxed state

equilibrium
state
no motion

system in
motion.
 $u(t)$ is displacement from eq.
Down is +,
up is -.

In the equilibrium state $F=0$. The force of the spring and the force of gravity are equal but opposite.

$$F = mg - kL = 0.$$

Thus $k = \frac{mg}{L}$.

This gives us a way to measure k if we know m , or m if we know k .

If the system is in motion we analyze the forces as follows,

$$F = mg - k(L + u(t)) = \underbrace{mg - kL}_{=0} - ku$$

But $F = mu''(t)$,

Thus $mu'' = -ku$ or

$$mu'' + ku = 0.$$

This is our model.

The characteristic equation ~~is zero~~

$$m r^2 + k = 0$$

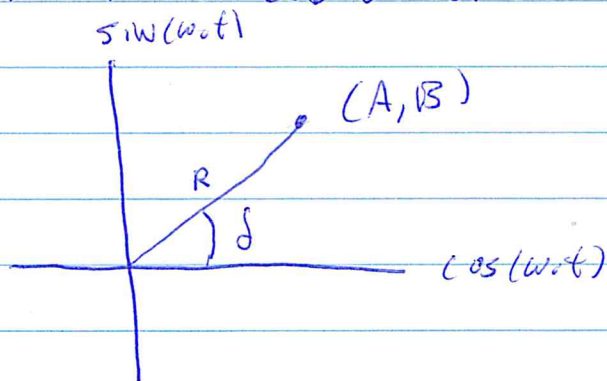
has root $r = 0 \pm i \sqrt{\frac{k}{m}}$. Let $\omega_0 = \sqrt{\frac{k}{m}}$.

Then the general solution is ↳ natural frequency

$$u(t) = A \cos(\omega_0 t) + B \sin(\omega_0 t)$$

It is standard practice to put this into a more useful form using some trig.

Think "polar coordinates".



$$R = \sqrt{A^2 + B^2}$$

$$\delta = \tan^{-1}(B/A) \text{ up to } \pi.$$

$$A = R \cos \delta$$

$$B = R \sin \delta$$

Now,

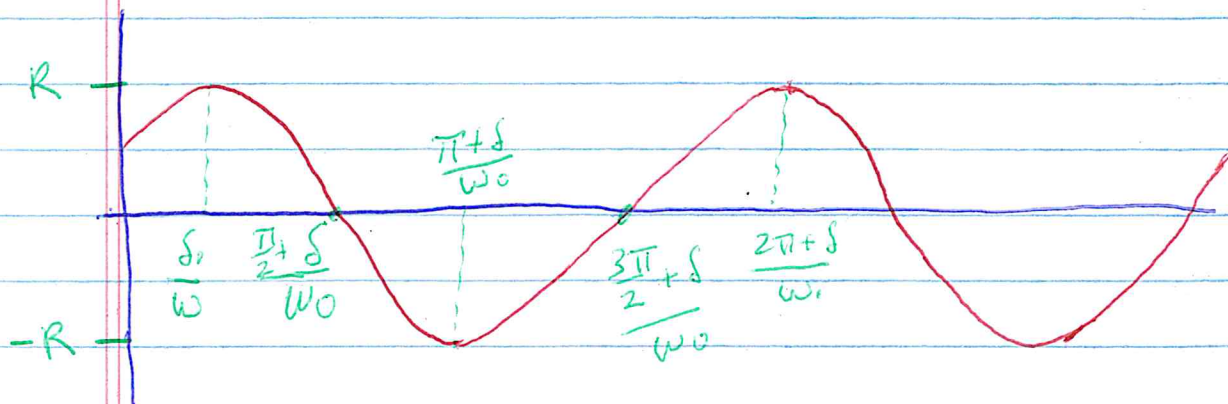
$$u(t) = R \cos \delta \cos \omega_0 t + R \sin \delta \sin \omega_0 t = R \cos(\omega_0 t - \delta)$$

amplitude ↗ ↗ phase shift

Notice,

$$u(0) = A = R \cos \delta$$

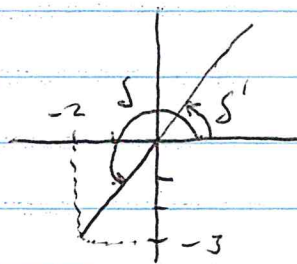
$$u'(0) = \omega_0 B = R \omega_0 \sin \delta$$



$$u(t) = R \cos(\omega_0 t - \delta)$$

Ex Suppose $u(t) = -2 \cos \pi t - 3 \sin \pi t$. Put into standard form.

Sol $R = \sqrt{4 + 9} = \sqrt{13}$, Let $\delta' = \tan^{-1}(\frac{-3}{-2}) = \tan^{-1}(\frac{3}{2})$
 $\approx 56.31^\circ$ or 0.9828 rad
 But $\delta = \delta' + \pi = 4.1244 \text{ rad}$ or 236.31° .



Thus $u(t) = \sqrt{13} \cos(\pi t - 4.1244)$

II Mass-spring systems with damping

We assume there is a force resisting the motion that is proportional to the velocity.

$$F_d = -\gamma u'(t)$$

γ is called the damping constant.

I'll let you figure out what its units might be.

Now we have $F = mu'' = mg - k(L+u) - \gamma u'$, or

$$\boxed{mu'' + \gamma u' + ku = 0.} \quad \left[\begin{array}{l} \text{Recall} \\ mg - kL = 0 \end{array} \right]$$

This is our model.

The characteristic equation is

$$mr^2 + \gamma r + k = 0.$$

$$\text{Its roots are } r = \frac{-\gamma \pm \sqrt{\gamma^2 - 4mk}}{2m}.$$

Our analysis is divided into 3 cases:

small damping $\gamma^2 - 4mk < 0$, large damping $\gamma^2 - 4mk > 0$
and critical damping $\gamma^2 - 4mk = 0$.

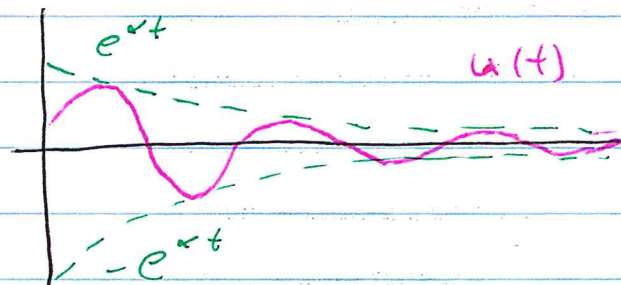
Case 1 Small damping, $\gamma^2 - 4mk < 0$.

Then $r = \alpha \pm i\beta$ where $\alpha = \frac{-\gamma}{2m}$, $i\beta = \frac{\sqrt{\gamma^2 - 4mk}}{2m}$

$$\text{or } \beta = \frac{\sqrt{4mk - \gamma^2}}{2m}.$$

$$\begin{aligned} \text{Then } u(t) &= A e^{\alpha t} \cos \beta t + B e^{\alpha t} \sin \beta t \\ &= e^{\alpha t} (A \cos \beta t + B \sin \beta t) = e^{\alpha t} R \cos(\beta t - \delta). \end{aligned}$$

Since $\alpha < 0$ the graph will look like



β is called the quasi frequency.

It is less than ω_0 , see page 199 eq (27).

Case 2 Large damping or overdamping, $\gamma^2 - 4mk > 0$.

Now the roots of $mr^2 + \gamma r + k = 0$ ~~and~~ are real and negative. We have

$$u(t) = A e^{r_1 t} + B e^{r_2 t}$$

$$u(t) \rightarrow 0 \text{ as } t \rightarrow \infty.$$

The initial conditions determine A and B.

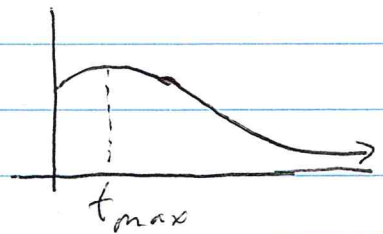
Sometimes it is useful to find the maximum u .

$$u' = A r_1 e^{r_1 t} + B r_2 e^{r_2 t} = 0$$

$$A r_1 + B r_2 e^{(r_2 - r_1)t} = 0$$

$$e^{(r_2 - r_1)t} = \frac{-A r_1}{B r_2}$$

$$t_{\max} = \frac{1}{r_2 - r_1} \ln \left(\frac{-A r_1}{B r_2} \right).$$



Then $u(t_{\max}) = \text{max value}$, if $t_{\max} \geq 0$.

Case 3 Critical Damping $\gamma^2 - 4mk = 0$.

Now $r = -\frac{\gamma}{2m}$ is the only root of the characteristic eq.

$$u(t) = A e^{rt} + B t e^{rt} = A e^{-\frac{\gamma t}{2m}} + B t e^{-\frac{\gamma t}{2m}}.$$

Show $t_{\max} = -\frac{rA+B}{rB}$.

Example (3.7 #9) A mass of 20 grams stretches a spring 5 cm. Suppose $\gamma = 400$ dyne-sec/cm. If the mass is pulled down 2 cm and then released, find $u(t)$. Express your answer as a cosine wave if the damping is small. Graph it. For what time T does $t \geq T \Rightarrow |u(t)| < 0.05$ cm?

Solution Step 1. Set up model.

$$m = 20 \text{ grams} \quad L = 5 \text{ cm} \quad \gamma = 400, \quad u(0) = 2, \quad u'(0) = 0.$$

$$\text{Find } k. \quad k = \frac{mg}{L} = \frac{20 \cdot 980 \text{ cm/s}^2}{5} = 3920.$$

Thus the model is

$$20 u'' + 400 u' + 3920 = 0, \quad u(0) = 2, \quad u'(0) = 0.$$

Step 2 Find general solution. The characteristic poly is

$$20r^2 + 400r + 3920 = 0$$

You can check that $r = -10 \pm i\sqrt{96}$. Thus,

$$u(t) = e^{-10t} (A \cos \sqrt{96}t + B \sin \sqrt{96}t).$$

Step 3 Find A and B. $u(0) = 2$, $u'(0) = 0$

$$u(0) = A \Rightarrow A = 2$$

$$u'(t) = -10e^{-10t} (\quad) + e^{-10t} (-A\sqrt{96} \sin \sqrt{96}t + B\sqrt{96} \cos \sqrt{96}t)$$

$$u'(0) = -10(A) + (B\sqrt{96})$$

$$= -20 + B\sqrt{96}$$

$$\Rightarrow B = \frac{20}{\sqrt{96}} \approx 2.0412414523$$

Step 4 Convert to cosine wave.

$$R = \sqrt{A^2 + B^2} = \frac{7}{\sqrt{6}} \approx 2.8577380333$$

$$\delta = \arctan\left(\frac{B}{A}\right) \approx 0.79560295355 \text{ radians.} \quad \left| \begin{array}{l} \text{no need to} \\ \text{add } \pi \end{array} \right.$$

Thus $u(t) \approx 2.858 e^{-10t} \cos(\sqrt{96}t - 0.7956)$

Plots for 3.7 #9
using Maple

```
> R:=7/sqrt(6);
```

$$R := \frac{7}{6} \sqrt{6}$$

```
> delta:=evalf(arctan(20/sqrt(96),2));
```

$$\delta := 0.7956029534$$

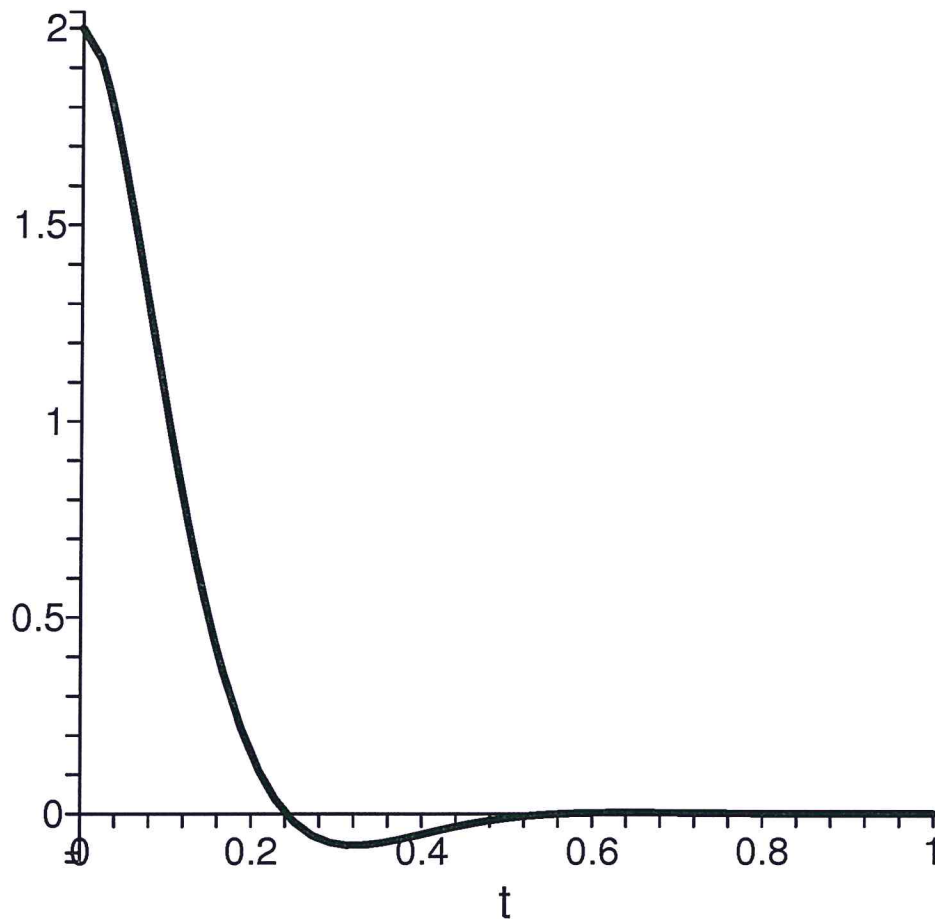
```
> beta:=sqrt(96);
```

$$\beta := 4 \sqrt{6}$$

```
> alpha:=-10;
```

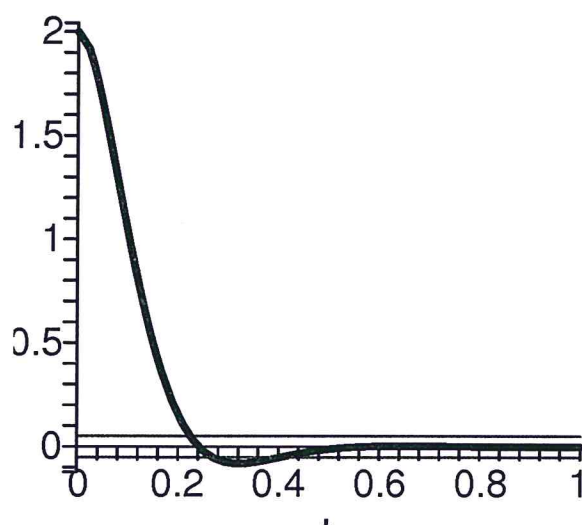
$$\alpha := -10$$

```
> plot(R*exp(alpha*t)*cos(beta*t - delta),t=0..1,color=black,thickness=2);
```



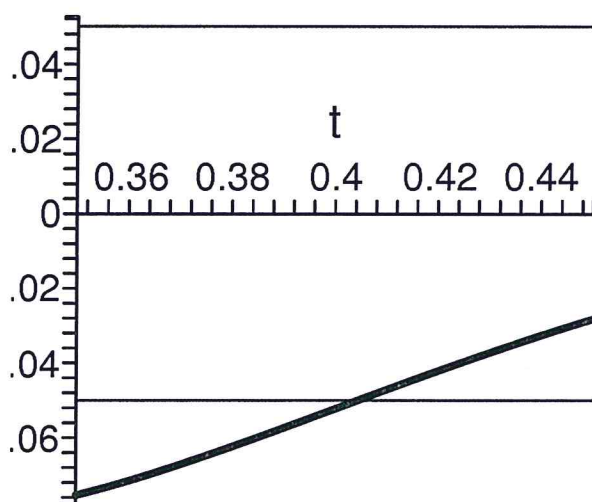
Find T for which $t > T$ implies $|u(t)| < 0.05$

```
> plot([-0.05,R*exp(alpha*t)*cos(beta*t -  
delta),0.05],t=0..1,thickness=[1,2,1],color=[red,black,red]);
```



We zoom in for a better look.

```
> plot([-0.05,R*exp(alpha*t)*cos(beta*t -  
delta),0.05],t=0.35..0.45,thickness=[1,2,1],color=[red,black,red]);
```



$T = 0.4045$, approximately.

We can get a better approximation using the solve command.

```
> solve(R*exp(alpha*t)*cos(beta*t - delta) = -0.05, t);  
0.4045411987
```

Example (3.7 #11) A spring is stretched 10 cm by a force of 3 N (Newtons). A mass of 2 kg is hung from the spring. When the velocity of the mass is 5 m/s a damping force of 3 N is measured. Now suppose the mass is pulled down 5 cm below equilibrium and given an initial velocity of 10 cm/s downward. Find a formula for the motion of this mass-spring system and graph it.

Solution Step 1, Set up the model.

$$m = 2$$

$$\gamma = 3/5$$

$$k = 3/0.1 = 30$$

$$u(0) = 5 \text{ cm} = 0.05 \text{ m}$$

$$u'(0) = 10 \text{ cm/s} = 0.1 \text{ m/s}$$

Thus our model is

$$2u'' + \frac{3}{5}u' + 30u = 0 \quad u(0) = 0.05, u'(0) = 0.1,$$

Step 2 Find gen. sol. I multiplied through by 5 so as not to have fractions.

$$10 u'' + 3u' + 150u = 0$$

$$10 r^2 + 3r + 150 = 0$$

$$r = \frac{-3 \pm \sqrt{9 - 4(6)(150)}}{20} = \frac{-3}{20} \pm \frac{i\sqrt{5991}}{20} \approx -0.15 \pm i3.8700775$$

Thus

$$u(t) = e^{-\frac{3}{20}t} \left(A \cos\left(\frac{\sqrt{5991}}{20}t\right) + B \sin\left(\frac{\sqrt{5991}}{20}t\right) \right)$$

Step 3 Find A and B. $u(0) = 0.05$, $u'(0) = 0.1$

$$u(0) = A \Rightarrow A = 0.05$$

$$u'(t) = -\frac{3}{20} e^{-\frac{3}{20}t} \left(\right) + e^{-\frac{3}{20}t} \left(-A \frac{\sqrt{5991}}{20} \sin\left(\frac{\sqrt{5991}}{20}t\right) + B \frac{\sqrt{5991}}{20} \cos\left(\frac{\sqrt{5991}}{20}t\right) \right)$$

$$u'(0) = -\frac{3}{20} A + B \frac{\sqrt{5991}}{20} = -0.6075 + B \frac{\sqrt{5991}}{20} = 0.1$$

$$\Rightarrow B = \frac{20}{\sqrt{5991}} (0.6075) \approx 0.02777722138$$

Step 4 Convert to cosine wave.

$$R = \sqrt{A^2 + B^2} \approx 0.057197675$$

$$\delta = \arctan\left(\frac{B}{A}\right) \approx 0.50709 \text{ rad. (No need to add } \pi)$$

$$u(t) \approx (0.057) e^{-0.15t} \cos(3.870t - 0.50709)$$

Plot for 3.7 #11
using Maple

```
> R:=sqrt((0.05)^2 + (2.15)^2/5991);  
R:= 0.05719767502
```

```
> delta:=evalf(arctan(43/sqrt(5991)));  
delta:= 0.5070900009
```

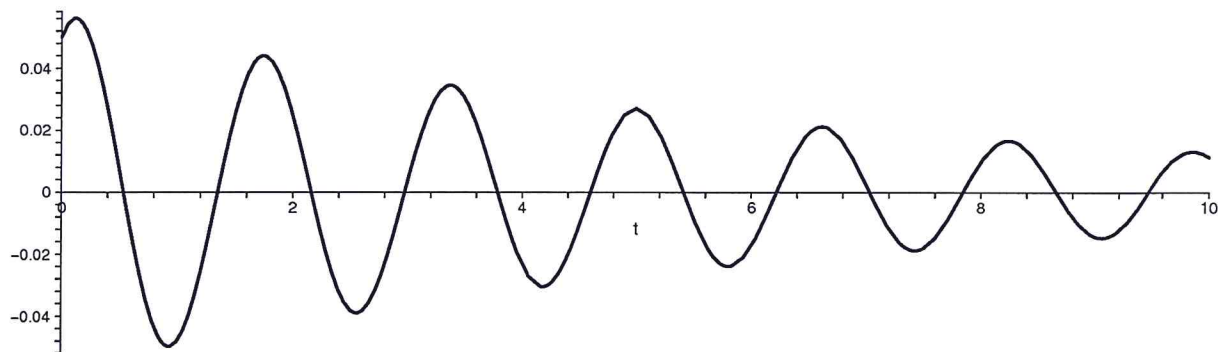
```
> beta:=sqrt(5991)/20;
```

$$\beta := \frac{1}{20} \sqrt{5991}$$

```
> alpha:=-3/20;
```

$$\alpha := \frac{-3}{20}$$

```
> plot(R*exp(alpha*t)*cos(beta*t - delta),t=0..10,color=black,thickness=2);
```



Example (3.7 #21) The Logarithmic Decrement

(a) For the damped oscillation described by Eq. (26), show that the time between successive maxima is $T_d = 2\pi/\mu$.

(b) Show that the ratio of the displacements at two successive maxima is given by $\exp(\gamma T_d/2m)$. Observe that this ratio does not depend on which pair of maxima is chosen. The natural log of this ratio is called the logarithmic decrement and is denoted by Δ .

Solution (a) Eq (26) is

$$u(t) = R e^{-\frac{\gamma t}{2m}} \cos(\mu t - \delta).$$

We are pretty sure T_d is just the period of $\cos(\mu t - \delta)$ which is $2\pi/\mu$, but we check.

Set $\frac{du}{dt} = 0$ and solve for t .

$$u'(t) = R \left(-\frac{\gamma}{2m}\right) e^{-\frac{\gamma t}{2m}} \cos(\mu t - \delta) - R \mu e^{-\frac{\gamma t}{2m}} \sin(\mu t - \delta) = 0.$$

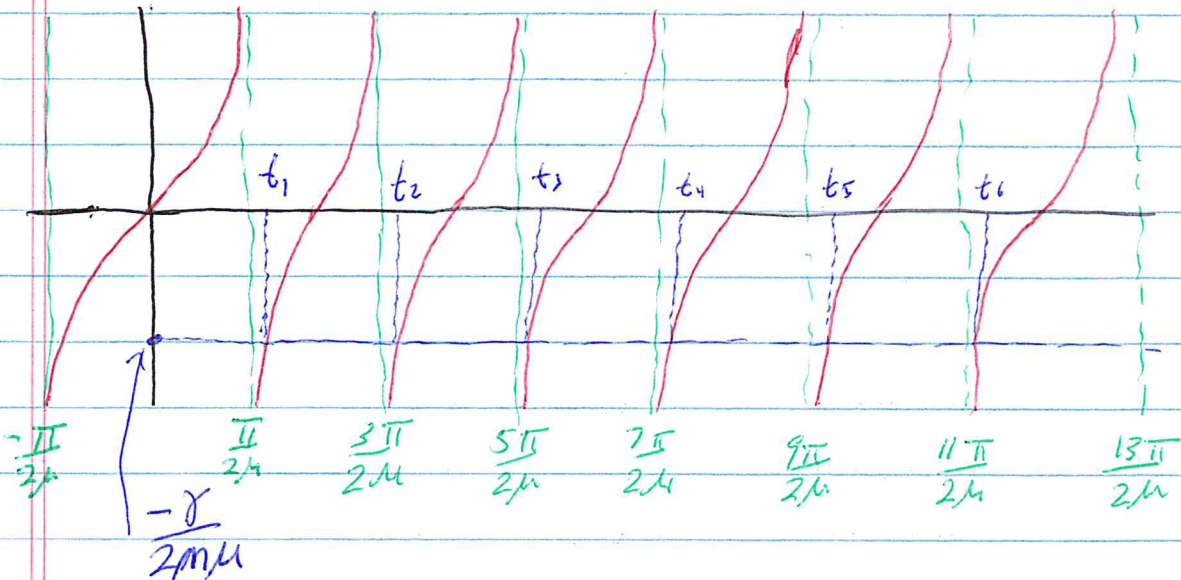
$$\text{Thus } \tan(\mu t - \delta) = \frac{-\gamma}{2m\mu} \quad (\text{As you will check!})$$

$$= X$$

Let $t_n = \frac{1}{\mu} \tan^{-1}\left(\frac{-\gamma}{2m\mu}\right) + \frac{\delta}{\mu} + n \frac{\pi}{\mu}$, $n = 1, 2, 3, \dots$

period of the tangent function

See the graph below to understand the idea ($\delta = 0$ for simplicity.)



Every other t_n corresponds to a maxima. Thus

$$T_d = t_{n+2} - t_n = \left(X + \frac{(n+2)\pi}{\mu}\right) - \left(X + \frac{n\pi}{\mu}\right) = \frac{2\pi}{\mu}$$

as we thought!

(b) We need to find $\frac{u(t_n)}{u(t_{n+2})}$.

We know $t_{n+2} = t_n + \frac{2\pi}{\mu}$.

$$u(t_n) = R e^{-\frac{\gamma t_n}{2m}} \cos(\mu t_n - \delta)$$

$$\begin{aligned} u(t_{n+2}) &= u\left(t_n + \frac{2\pi}{\mu}\right) = R e^{-\frac{\gamma t_n}{2m} - \frac{2\gamma\pi}{2m\mu}} \cos\left(\mu\left(t_n + \frac{2\pi}{\mu}\right) - \delta\right) \\ &= R e^{-\frac{\gamma t_n}{2m}} e^{-\frac{2\gamma\pi}{m\mu}} \cos(\mu t_n + 2\pi - \delta) \\ &= \text{" " " } \cos(\mu t_n - \delta). \end{aligned}$$

$$\frac{u(t_n)}{u(t_{n+2})} = \frac{R e^{-\frac{\gamma t_n}{2m}} \cos(\mu t_n - \delta)}{R e^{-\frac{\gamma t_n}{2m}} e^{-\frac{2\gamma\pi}{m\mu}} \cos(\mu t_n - \delta)} = e^{\frac{2\gamma\pi}{m\mu}}$$

$$\text{Since } T_d = \frac{2\pi}{\mu}, \quad e^{\frac{2\gamma\pi}{m\mu}} = \boxed{e^{\frac{\gamma T_d}{2m}}} \quad \text{Victory!!}$$

Clearly the result is independent of n .

$$\text{Let } \Delta = \ln\left(e^{\frac{\gamma T_d}{2m}}\right) = \frac{\gamma T_d}{2m}.$$

$$\text{Then } \gamma = \frac{2m\Delta}{T_d}. \quad \text{Each of } m, \Delta \text{ and } T_d$$

is easy to measure in an experiment. Thus we have an experimental means of find the damping constant. That is the point of this problem.